

REMARKS

Claims 1-6, 9, 10, 12, 13, 16-21, 24, 25, 29 and 30 are pending in this application. By this Amendment, claims 1-6, 9, 10, 12, 16-21, 24, 25 and 30 are amended. Claims 7, 8, 11, 14, 15, 22, 23, 26-28 and 31-34 are canceled without prejudice to or disclaimer of the subject matter recited therein.

Applicants appreciate the indication of allowable subject matter in claims 3, 7, 10 and 18-20, they being allowable if rewritten to overcome the rejection under 35 U.S.C. §112, second paragraph, and to include all of the features of their base claim and any intervening claims. However, Applicants assert that none of the allowable claims are rejected under 35 U.S.C. §112, second paragraph. Rather, only claim 15 is rejected on that basis. Furthermore, each of claims 3, 7, 10 and 18-20 depend directly or indirectly from claim 1. As stated above, the subject matter of allowable claim 7 is incorporated into claim 1. Accordingly, all pending claims are in condition for allowance.

The Drawings

The November 8, 2002 Office Action disapproves of the proposed drawing corrections filed on August 1, 2002. Therefore, the drawings remain objected to under 37 C.F.R §1.83(a) for showing features of the invention not specified in the claims. Specifically, features of claims 8, 11, 12, 14, 22, 23, 26, 27, and 31-34 are alleged not to be shown in the drawings. As claims 8, 11, 14, 22, 23, 26, 27 and 31-34 are canceled, the objection to the drawings under 37 C.F.R §1.83(a) regarding the features recited therein is moot. Applicants traverse the objection to the drawings regarding the features shown in claim 12.

Claim 12 recites *inter alia*, "wherein the cylinder space is formed in the nut". Fig. 2 clearly shows the cylinder space 38 and the nut 14. Accordingly, Applicants request the objection to the drawings be withdrawn.

The Abstract

The Office Action alleges that the Abstract accompanying the Patent Cooperation Treaty Application contains legal language such as "means". As an Abstract has not been filed for the U.S. Application, Applicants submit the Abstract attached hereto which does not contain the objected to language.

The Specification

The disclosure is objected to for lacking section headings. The disclosure is amended in the substitute specification attached hereto.

The application is also objected to because of "hand written" alterations in the copy of the specification received by the U.S. Patent and Trademark Office from the International Bureau. A substitute specification is included in reply to the objection.

The Office Action further requires a properly executed Oath and Declaration which complies with 37 C.F.R §1.67(a). Applicants assert that a Declaration fully executed by the inventors was filed on February 23, 2001. Notification of acceptance of the Oath and Declaration was mailed by the U.S. Patent and Trademark Office on March 3, 2001. Accordingly, Applicants assert that a properly executed Oath and Declaration that complies with 37 C.F.R §1.67(a) has been filed and accepted in the U.S. Patent and Trademark Office. Accordingly, Applicants assert that the requirement of a new Declaration is improper.

Rejection Under 35 U.S.C. §112

Claims 8, 11, 12, 14, 22, 23, 26-28, and 31-34 are rejected under 35 U.S.C. §112, first paragraph. As claims 8, 11, 14, 22, 23, 26-28 and 31-34 are canceled the rejection of those claims is moot. Applicants respectfully traverse the rejection of claim 12.

Applicants assert that claim 12 contains a subject matter that is adequately supported in the specification to enable one skilled in the art to which it pertains to make and/or use the invention. Claim 12 recites an actuator according to claim 9, wherein the cylinder space is

formed in the nut. Support for the recitation may be found in the specification at least at page 3, line 31 through page 4, line 2. Furthermore, Fig. 2 clearly shows the cylinder space and the nut. Accordingly, Applicants request rejection claim 12 under 35 U.S.C. §112, first paragraph, be withdrawn.

Claim 15 is rejected under 35 U.S.C. §112, second paragraph. Claim 15 is canceled and therefore the rejection of that claim is moot.

Rejection Under 35 U.S.C. §102

Claims 1, 2, 4, 9, 13, 21, 25, 29 and 30 are rejected under 35 U.S.C. §102(b) as anticipated by U.S. Patent 5,788,341 (to Penrod et al.). The rejection is respectfully traversed.

Applicants assert that Penrod does not disclose each and every feature recited in the amended claims. For example, the subject matter recited in allowable claim 7 is incorporated into independent claim 1. Accordingly, independent claim 1 and dependent claims 2, 4, 9, 13, 21, 25, 29 and 30 recite subject matter not disclosed in Penrod. Thus, Applicants respectfully request the rejection of claims 1, 2, 4, 9, 13, 21, 25, 29 and 30 under 35 U.S.C. §102(b) be withdrawn.

Rejections Under 35 U.S.C. §103

Claims 1, 2, 4-6, 8, 9, 13, 16, 17, 24, 25, 29 and 30 are rejected under 35 U.S.C. §103(a) as unpatentable over WO 97/17553 in view of U.S. Patent 4,850,457 to Taig. As claim 8 is canceled the rejection of that claim is moot. Applicants respectfully traverse the rejection of claims 1, 2, 4-6, 9, 13, 16, 17, 24, 25, 29 and 30.

As stated above, the subject matter recited in allowable claim 7 is incorporated into independent claim 1. Accordingly, each of the pending rejected claims includes allowable subject matter. Thus, Applicants respectfully request rejection of the rejected claims be withdrawn.

Claim 15 is rejected under 35 U.S.C. §103(a) as unpatentable over Penrod in view of U.S. Patent 6,012,556 to Blosh et al. As claim 15 is canceled rejection of that claim is moot.

Claims 16 and 17 are rejected under 35 U.S.C. §103(a) as unpatentable over Penrod in view of Taig. The rejection is respectfully traversed.

Applicants assert that claims 16 and 17 are allowable for the at least dependency on independent claim 1 for the reasons discussed above, as well as for the additional features recited therein. Thus, Applicants respectfully request rejection of claims 16 and 17 under 35 U.S.C. §103(a) be withdrawn.

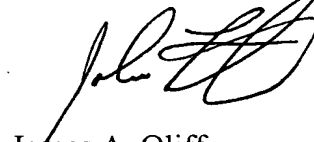
Claim 24 is rejected under 35 U.S.C. §103(a) as unpatentable over Penrod. The rejection is respectfully traversed.

Applicants assert that claim 24 is allowable for at least its dependency on independent claim 1 for the reasons discussed above, as well as for the additional features recited therein. Accordingly, Applicants respectfully request rejection of claim 24 under 35 U.S.C. §103(a) be withdrawn.

In view of the foregoing, it is respectfully submitted that this application is in condition for allowance. Favorable reconsideration and prompt allowance of claims 1-6, 9, 10, 12, 13, 16-21, 24, 25, 29 and 30 are earnestly solicited.

Should the Examiner believe that anything further would be desirable in order to place this application in even better condition for allowance, the Examiner is invited to contact the undersigned at the telephone number set forth below.

Respectfully submitted,



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Attachments:

Abstract of the Disclosure
Substitute Specification
Marked-Up Copy of Specification

Date: July 8, 2003

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SCREW ACTUATOR, AND BRAKE CALIPER COMPRISING SUCH ACTUATOR

BACKGROUND OF THE INVENTION

1. Field of Invention

2. Description of Related Art

[0001] The invention is related to a screw actuator, comprising an actuating member and a screw mechanism having a screw, nut and rolling elements, one of which screw and nut is rotatably supported with respect to the housing and providing a linear movement of the actuating member with respect to the housing in response to a rotational movement of the motor, and a reduction gear means.

2. Description of Related Art

[0002] Such screw actuator is known from ~~WO-A-960331~~GB-A-2291949. Said known screw actuator comprises a screw mechanism which is supported with respect to the housing by means of a bearing capable of accommodating axial and/or radial loads, e.g. an axial thrust bearing for accommodating the axial forces exerted on the brakepads.

[0003] This screw mechanism is a so-called roller screw mechanism. Depending on the application condition constraints i.e. space available, and load, one can select a roller screw or a ball screw type actuator. Specific application considerations for a roller screw is that such roller screw mechanism provides a high power density, which means that within specific dimensional constraints, a relatively high load carrying capacity can be provided. Said carrying capacity however is predominantly related to axial loads. With respect to radial loads, the carrying capacity is less favourable compared to a ball screw. A roller screw mechanism is in general more sensitive with respect to radial loads and misalignment.

[0004] Another specific component in a roller screw mechanism is the cage which is necessary to space the rollers. In high speed applications this cage mass results in higher starting torques.

SUMMARY OF THE INVENTION

[0005] The object of the invention is to provide an improved actuator. This object is achieved in that the nut is fixed with respect to the housing, and the screw is rotatably supported with respect to the housing by means of the rolling elements. Said rolling elements may comprise rollers or balls.

[0006] In order to obtain about the same load bearing capacity as in a roller screw mechanism, the pitch diameter of the rolling balls, the ball diameter and its contact angle with screw and nut, and the number of turns should be designed such that appropriate dimensions

and the required load carrying capacity are provided. However, as the rolling elements of the screw mechanism can also act as bearing elements for supporting the rotating screw, no separate bearing is necessary to take up the axial load. As a result, the overall dimensions for a ball screw mechanism may remain limited in order to meet application requirements.

[0007] The ball screw mechanism is less sensitive with respect to radial loads, which makes it less vulnerable for misalignments. Also, no cage is needed for recirculation of the balls. Instead, recirculation of the balls may be obtained by means of recirculation tube or hole plug between the first and the last ball row or for each ball turn.

[0008] The axial moving and rotating screw according to the invention may be driven by the reduction gear means through a coupling means which allows axial displacements. Said coupling means may comprise a drive shaft accommodated within a bore in the screw, the surface of the drive shaft and the bore having axial grooves which engage each other through balls or splines.

[0009] The screw may engage the actuating member through a bearing capable to cope with radial and axial load in order to stop the rotating motion in relation to the moving actuating member.

[0010] The reduction gear means is preferably contained in a reduction gear module and the screw mechanism is contained in a screw mechanism module.

[0011] The actuating member may be executed as a piston, which is slidably held within a cylinder space of the housing. Said piston can be held non-rotatably by means of a groove and pin assembly. The motor drive module can be mounted in-line with the actuator or in angled position.

[0012] For a right angle position of the motor module, the reduction gear means may comprise one or more reduction steps with at least part of a planetary gear system having a stationary outer ring gear wheel with inwardly pointing gear teeth. In particular, the reduction gear means may comprise satellite gear wheels which mesh with the ring gear wheel and which are accommodated on a carrier connected to a rotary shaft engaging the screw mechanism, and the sun gear wheel of the planetary gear system may be accommodated on a drive shaft of the drive module. This system provides an optimal axial compactness of the application.

[0013] The sun gear wheel of the reduction gear means is connected to an angled or right angle gear reduction e.g. a bevel gear which mates with a motor driven bevel pinion. Said sun gear wheel and the bevel gear are carried out as a unitary gear wheel which is supported with respect to the nut of the screw mechanism by means of a rolling element

bearing. In order to achieve an appropriate reduction, the pitch diameter of the bevel gear is larger than the pitch diameter of the sun gear wheel.

[0014] Furthermore, a sensor fixed on a bearing or near the motorshaft may be provided for detecting rotational and/or translational movements of the screw mechanism or other operating parameters. Also, control means may be provided, said control means having an input for a control signal, e.g. from a brake pedal, and being connected to the sensor for controlling the electric motor on the basis of the control signal and the signal from the sensor. The sensor is in particular suitable for obtaining force feedback, wear compensation and/or maintenance indication.

[0015] The actuator according to the invention can be applied for different purposes. In particular, the actuator is suitable for use in a brake caliper for an electrically actuatable disc brake, said caliper comprising an actuator as described before, and a claw piece carrying two opposite brake pads, said actuator comprising a screw and a nut one of which is rotatably supported with respect to the housing by means of an angular bearing, and a reduction gear means.

[0016] The invention will further be described with reference to the embodiments of figures 1 and 2.

BRIEF DESCRIPTION OF THE DRAWINGS

[0017] Figure 1 shows a brake caliper, comprising an actuator according to the invention, in exploded view.

[0018] Figure 2 shows the brake caliper according to figure 1, in assembled state.

[0019] Figure 3 shows a detail.

[0020] Figure 4 shows a detail of the ball screw.

~~[0021] Figure 5 shows a further embodiment.~~

DETAILED DESCRIPTION OF PREFERRED EMBODIMENTSINVENTION

[0021] The brake caliper shown in figures 1 and 2 comprises a claw piece 1 carrying a fixed brake pad 2 and a displaceable brake pad 3. Said brake pads 2, 3 can be brought into co-operation with brake disc 4.

[0022] The displaceable brake pad 3 engages a ball screw mechanism 5 which by means of reduction gear means 6 is driven by motor 7. Said motor 7 may be provided with a sensor 40, connected to the motor shaft.

[0023] More in particular, the displaceable brake pad 3 is connected by means of bolt 8 and screwthreaded hole 9 to an actuating member 10. Said actuating member 10 engages the screw 11 by means of an a bearing 12 capable to take up axial load. Said

actuating member is carried out as a piston 10, which is slidably, but non-rotatably held in a cylinder space 38 in the housing 17.

[0024] By means of balls 13, screw 11 engages the nut 14. Said nut 14 has an external screwthread 15, by means of which the nut 14 is connected to the housing 17. Moreover, a recirculating tube 18 for recirculating the balls 13 upon rotating the screw 11 with respect to said nut 14, extends through the nut.

[0025] The screw 11 has a bore 37 with internal grooves 19, which engage balls 20. Said balls 20 also engage the external grooves 21 of drive shaft 22.

[0026] By rotating drive shaft 22 through reduction gear means 6 and motor 7, the screw 11 is rotated as well. As a result, it is displaced backward or forward by the co-operation of its screw type groove 23 with the screw type groove 24 of the nut 14, by means of the balls ~~15~~13.

[0027] Drive shaft ~~23~~22 is connected to a carrier 25, which carries satellite gear wheels 26. Said satellite gear wheels 26 each engage a ring gear wheel 27 as well as a sun gear wheel 28.

[0028] Sun gear wheel 28 forms a unity with bevel gear 29 which together form a unitary gear wheel 30. Said unitary gear wheel 30 by means of bearing 31 is supported with respect to the nut 14.

[0029] The bevel gear 29 engages the bevel pinion ~~31~~32, which in turn is driven by motor 7.

[0030] The bearing 31, which supports the unitary gear wheel with respect to the nut 14, comprises a sensor 33 for detecting the rotations of the screw mechanism, and thereby the displacement of the displaceable brake pad 3.

[0031] Housing 17 comprises a bore 34, through which a wire can be guided to the outside from said sensor 33.

[0032] The carrier 25 is supported with respect to the housing 17 by means of bearing 35; by means of bolts 36, motor 7 is connected to said housing 17.

[0033] According to the detail of figure 3, the connection between brake pad 3 and piston 10 may alternatively be obtained through edges~~43~~ which are slidable mounted in grooves 43 of piston 10.

[0034] In order to accommodate the axial forces exerted on the ball screw mechanism when applying a brake force on the brake pads 2, 3, the screw threads 23, 24 of screw 11 respectively nut 14 can be adapted according to the embodiment shown in figure 4

In cross-section, the threads 23 have raised parts 51, whereas the threads 24 have raised parts 52.

[0035] As a result of these shapes, the working lines 50 as defined by the contact angles and the ball conformity with the ball tracks, which define the load paths which play a role in the force transfer, are in a more inclined position. The contact angle is between 45-70° in order to create optimized load carrying capacity for the ball screw in relation to the applied load specification.

~~—— [0036] The embodiment of figure 5 comprises a motor 61, having a stator 53 and a rotor 54 connected to a sleeve 55. The sleeve 55 is connected through the intermediate piece 56 to drive shaft 22. Alternatively, the sleeve 55 may be connected to the drive shaft 22 through a gear reduction.~~

~~—— [0037] Drive shaft 22 drives screw 11 of screw mechanism 5, through the groove 21, 19 in the respectively the drive shaft 22 and the screw 11, as well as through the balls 20 accommodated in said grooves 21, 19.~~

~~—— [0038] Via thrust bearing 69, the screw 11 can be connected to e.g. a brake pad in case of an actuator applied in a claw piece. The thrust bearing 69 comprises two rings 67, 68, and balls 12. One of the rings 67 may form a unity with the screw 11. The other ring 68 may comprise a locking groove 66 for locking a brake pad (not shown) thereto.~~

~~—— [0039] The screw 11 is rotatably and translatably held in a cylinder space 59 defined by insert ring 58 inserted in nut 14 of the screw mechanism 5. Nut 14 and screw 11 of screw mechanism 5 engage each other through balls 13, accommodated in respective screwthreaded threads of nut 14 and screw 11.~~

~~—— [0040] Sleeve 55 connected to rotor 54 of motor 61 is rotatably supported on the fixed nut 14 by means of bearings 63. These bearing have an outer race accommodated in the sleeve 55, and an inner ring 57 having appropriate raceways as well. The inner ring 57 is locked by means of locking ring 65. Furthermore, these bearing 63 have balls 64.~~

~~—— [0041] Alternatively, two separate standard ball or roller bearings may be applied.~~

~~—— [0042] The housing of the actuator is indicated by reference number 62.~~